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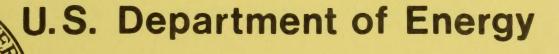
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# Monthly Performance Report

BLAKEDALE PROFESSIONAL CENTER
APRIL 1979





National Solar Heating and Cooling Demonstration Program

National Solar Data Program

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# MONTHLY PERFORMANCE REPORT BLAKEDALE PROFESSIONAL CENTER APRIL 1979

### I. SYSTEM DESCRIPTION

The Blakedale Professional Center solar energy system is designed to provide 85 percent of the space heating requirements and 100 percent of the domestic hot water heating requirements for a 4,400 square foot office suite in Greenwood, South Carolina. Solar energy is collected by 53 flat-plate collectors, which are manufactured by PPG Industries. The collectors, having a gross area of 954 square feet, are arranged in three banks and are mounted on the roof. Each collector array faces south at an angle of 45 degrees from the horizontal. The heat transfer medium is 99 percent water and one percent corrosion inhibitor. Solar energy is stored in a 5,000-gallon tank that is buried under the parking lot. The tank is insulated with four inches of sprayed-on polyurethane which is covered with a water-proof coating. When solar energy is inadequate, auxiliary space heating is provided by a 10-ton heat pump and a 36-kilowatt electric resistance heater. Auxiliary hot water heating is provided by an electrical element in a 40-gallon tank. Freeze protection is provided by a drain-down system.

The system, shown schematically in Figure 1, has four modes of operation:

Mode 1 - Collector-to-Storage: This mode is entered when the difference between the temperature of the collector and the temperature of water near the bottom of the water thermal storage is greater than 19°F. Pump Pl circulates water through the collectors to transfer solar energy to the water thermal storage. This mode terminates when the temperature differential is less than 6°F, or the temperature of water in the collector is less than 37°F.

<u>Mode 2 - Storage-to-Office Area (Solar)</u>: This mode is entered when heat is required in the office area. Pump P2 circulates water through the water thermal storage to heat exchanger HX2 in the air-handling unit. This mode

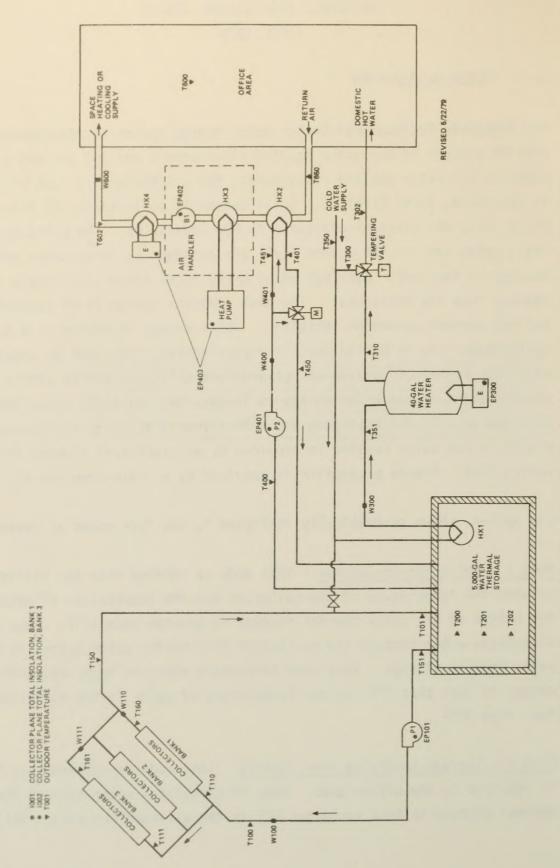


Figure 1. BLAKEDALE PROFESSIONAL CENTER SOLAR ENERGY SYSTEM SCHEMATIC

terminates when the supply air temperature is greater than 120°F, as the circulation bypasses this heat exchanger by the action of the motorized valve, or the requirement for heat is satisfied.

Mode 3 - Storage-to-Office Area (Auxiliary): Although this mode is not a solar mode of operation, it is entered concurrently with Mode 2 when heat is required in the office and the office return air temperature is less than 65°F. A 10-ton heat pump is energized to provide thermal energy to heat exchanger HX3. When the outside air temperature is less than 40°F, two 18-kilowatt electric resistance heaters are energized in stages to provide auxiliary energy to heat exchanger HX4. This mode terminates when the office return air temperature is greater than 68°F, or the requirement for heat is satisfied.

<u>Mode 4 - Domestic Hot Water Preheating</u>: This mode is entered when there is a requirement for hot water. As hot water is drawn, cold water passes through heat exchanger HXl in the water thermal storage, and subsequently through the water heater and tempering valve to provide 120°F water. This mode terminates when the requirement for hot water is satisfied.

### II. PERFORMANCE EVALUATION

The system performance evaluations discussed in this section are based primarily on the analysis of the data presented in the attached computer-generated monthly report. This attached report consists of daily site thermal and energy values for each subsystem, plus environmental data. The performance factors discussed in this report are based upon the definitions contained in NBSIR 76-1137, Thermal Data Requirements and Performance Evaluation Procedures for the National Solar Heating and Cooling Demonstration Program.

### A. Introduction

The Blakedale Professional Center solar energy system was operational for the first 20 days in April. After April 20, the solar energy system was shut down for the summer as the heating season was essentially completed. There were some minor space heating requirements after April 20 but these were satisfied by the heat pump auxiliary system.

Since the solar energy system did not operate after April 20, some of the daily averages presented in the attached computer generated monthly report will be in error. This is because the daily averages are based on the full 30-day month, rather than the fraction of the month that the solar energy system was operational. The daily averages that will be affected are those that primarily relate to the operation of the solar energy system. These would include such things as solar energy collected, collector array efficiency, various operating energies, most of the storage subsystem parameters and all space heating subsystem parameters that are based on some solar energy contribution.

During April, the solar energy system satisifed 52 percent of the total space heating load. However, the solar fraction was 58 percent for the first 20 days of the month. The net electrical energy savings for the space heating subsystem were 0.10 million Btu, but, overall, the system operated at a net loss of 0.13 million Btu for the month.

During April, there were some control system problems that resulted in anomalous operation of the pumps (Pl and P2) in the solar energy system. These problems were discussed in detail in the March report, and they result essentially in an unnecessary waste of both solar and auxiliary energy. Unscheduled operation of pump Pl resulted in 0.23 million Btu of solar energy being removed from storage and lost through transport losses or by rejection from the collector array. During periods when the solar portion of the space heating subsystem was operating (9 of the first 20 days in the month), excessive operation of pump P2 caused an excess consumption of 0.40 million Btu of operating energy and a loss of approximately 0.75 million Btu of solar energy.

### B. Weather

During April, the measured average outside ambient temperature was  $66^{\circ}F$ . This was four degrees above the long-term average ambient temperature of  $62^{\circ}F$  and reduced the building heating load. The measured value for insolation incident in the plane of the collector array was 1,424 Btu/ft²-day, and this was 12 percent below the long-term average of 1,625 Btu/ft²-day. Long-term average temperature data and long-term average insolation data are taken from Reference Monthly Environmental Data for Systems in the National Solar Data Network, Department of Energy Report SOLAR/0019-79/36.

### C. System Thermal Performance

<u>Collector</u> - During the reporting period, a total of 40.76 million Btu of incident solar energy was measured in the plane of the collector array. The system collected a net amount of 6.42 million Btu, or 16 percent of the total available insolation. During the time when the system was operating, a total of 22.29 million Btu of solar energy was incident on the array.

This represents an operational collector efficiency of 29 percent. A total of 0.23 million Btu of electrical energy was required to operate the collector array. However, part of this operating energy expenditure was due to unscheduled operation of pump Pl, and this allowed 0.23 million Btu of stored solar energy to be rejected from the collector array to the atmosphere.

Storage - During April, a total of 6.60 million Btu was delivered to the storage tank. However, the unscheduled operation of pump Pl caused 0.60 million Btu to be removed from storage and lost, either through the collector array (0.23 million Btu) or the system piping (0.37 million Btu). Therefore, there was a net amount of 6.00 million Btu delivered to storage during the month.

A total of 2.62 million Btu was removed from storage during the reporting period. Since 1.85 million Btu was actually delivered to the space heating subsystem, there were 0.77 million Btu in transport losses. The significant point regarding these transport losses is that the majority of them (0.75 million Btu) resulted from the excessive operation of pump P2.

The change in stored energy was 0.21 million Btu and this, combined with the energy supplied to and removed from storage, resulted in a thermal loss from storage of 3.17 million Btu. Storage was maintained at an average temperature of 118°F, and the monthly storage efficiency was 0.47.

Hot Water Load - The hot water load at the Blakedale Professional Center is very small. The make-up water is heated by the storage tank and the electrical auxiliary system is not used. Therefore, although the solar energy system carries the entire hot water load, this load is small enough to be neglected.

Space Heating Load - The space heating load for April was 3.55 million Btu. However, this value represents the load for the entire month. During the period when the solar energy system was active, the total space heating load was 3.17 million Btu and solar energy satisfied 1.85 million Btu or 58 percent of this load. The remainder of the energy required for space heating was delivered by the heat pump.

As noted in the storage section, 0.75 million Btu of solar energy were wasted by the space heating subsystem due to excessive operation of pump P2. This represents 41 percent of the total amount of solar energy that was delivered to the space heating load.

During April, the total amount of energy delivered to the space heating subsystem maintained an average building temperature of 73°F. The average outdoor ambient temperature was 66°F.

### D. Observations

The April operation of the Blakedale Professional Center solar energy system was somewhat degraded due to the control problems previously discussed. This installation has a relatively large energy collection and storage subsystem (ECSS), when compared to the actual size of the building, and a higher level of performance should be expected from this solar energy system.

### E. Energy Savings

During April, the space heating subsystem realized a net savings of only 0.10 million Btu. However, when the 0.23 million Btu of electrical energy required to operate the ECSS is considered, then the overall system actually operated at a loss of 0.13 million Btu for the month. Part of this loss was due to the unscheduled operation of pump Pl, but the majority of the loss came from the excessive operation of pump P2. This pump consumed 0.40 million Btu of electrical energy by operating during periods when no solar energy was being supplied to the space heating subsystem. If the system had been operating properly, the net savings for April would have been at least 0.27 million Btu.

### III. ACTION STATUS

The system contractor is aware of the existing control problems. However, it is not known at the present time what action will be taken to correct them prior to the start of the next heating season.

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